The background features a central globe with the ANSYS logo overlaid. The globe is surrounded by a complex network of glowing blue and orange lines that radiate outwards, creating a sense of energy and motion. The ANSYS logo is prominently displayed in the center of the globe.

Numerical Prediction of Stress Evolution During Cooling of Glass Containers Using the Narayanaswamy Model

Benoît Debbaut and Thierry Marchal,
ANSYS / Polyflow s.a., Wavre, Belgium
Matt Hyre,
Emhart Glass Research Center, Windsor, CT, USA

- Introduction
- Business drivers
- Modelling
- *Validation vs. ANSYS*
- Brown glass
- Bottle
- Conclusions

- **Glass forming: a delicate process to manufacture containers**
 - Complex thermal phenomena
 - Large deformations of the glass domain during the process
 - Multiphase challenge: solid devices, molten glass, air cooling
- **Motivation for engineering simulation solution**
 - Virtual prototyping for the whole manufacturing process
 - Calculation of the flow pattern, deformation of the free surface, temperature field
 - Prediction of weight variation, stress induced during processing and cooling, possible defect
 - Improvement of existing process and cost-effective innovations

- **Increasing competitive level for containers manufacturing**
 - Globalization of the production market
- **Production of innovative glass containers with**
 - Thinner wall
 - More complex geometries
 - Different materials
- **Reduction of product cycles**
- **Target defect-free production**
- **Emergence of Simulation Driven Product Design**
 - Modeling of the forming, cooling and testing processes

- **Calculated quantities**

- Displacement \underline{u}
- Deformation tensor $\underline{\underline{\varepsilon}}$
- Stress tensor $\underline{\underline{\sigma}}$
- Thermal strain Θ
- Temperature T
- Fictive temperature T_f
- Reduced time ξ

- **Data**

- Bulk modulus $K(t) = K_{\infty} + \sum_{p=1}^P K_p \exp[-t/\tau_p]$

- Shear modulus $G(t) = G_{\infty} + \sum_{q=1}^Q G_q \exp[-t/\lambda_q]$

- Structural modulus $M(t) = \sum_{r=1}^R M_r \exp[-t/\mu_r]$

- Activation energy A

- Structural parameter x

- Reference temperature T_{ref}

- Density ρ , thermal conductivity κ , heat capacity C_p

- Liquid and glassy dilatation coefficients α_l and α_g

- **Equations (i)**

- Deformation tensor

$$\underline{\underline{\varepsilon}} = \frac{1}{2} [\nabla \underline{u} + (\nabla \underline{u})^T]$$

- Momentum equation

$$\nabla \cdot \underline{\underline{\sigma}} = 0$$

- Stress constitutive equation

$$\underline{\underline{\sigma}}(t) = I \int_0^t K[\xi(t) - \xi(\tau)] \frac{d}{d\tau} (tr(\underline{\underline{\varepsilon}}) - \Theta) d\tau + 2 \int_0^t G[\xi(t) - \xi(\tau)] \frac{d}{d\tau} \left(\underline{\underline{\varepsilon}} - \frac{I tr(\underline{\underline{\varepsilon}})}{3} \right) d\tau$$

- **Equations (ii): Narayanaswamy model**

- Reduced time variable

$$\xi(t) = \int_0^t \exp \left[Ax \left[\frac{1}{T_{ref}} - \frac{1}{T(\tau)} \right] + A(1-x) \left[\frac{1}{T_{ref}} - \frac{1}{T_f(\tau)} \right] \right] d\tau$$

- Fictive temperature

$$T_f(t) = T(t) - \int_0^t M[\xi(t) - \xi(\tau)] \frac{dT(\tau)}{d\tau} d\tau$$

- Thermal strain

$$\Theta = \int_{T_0}^{T_f} a_l(u) du + \int_{T_f}^T a_g(u) du$$

- **Interpretation of the Narayanaswamy model**
 - Stress build up:
 - non-uniform cooling
 - -> non-uniform distribution of fictive temperature
 - -> non-uniform distribution of thermal strain
 - Slow cooling, T_f is close to T ,
Low residual stress
 - Fast cooling, T_f can be “frozen” at high values,
Large residual stress
 - Relaxation mechanism in time

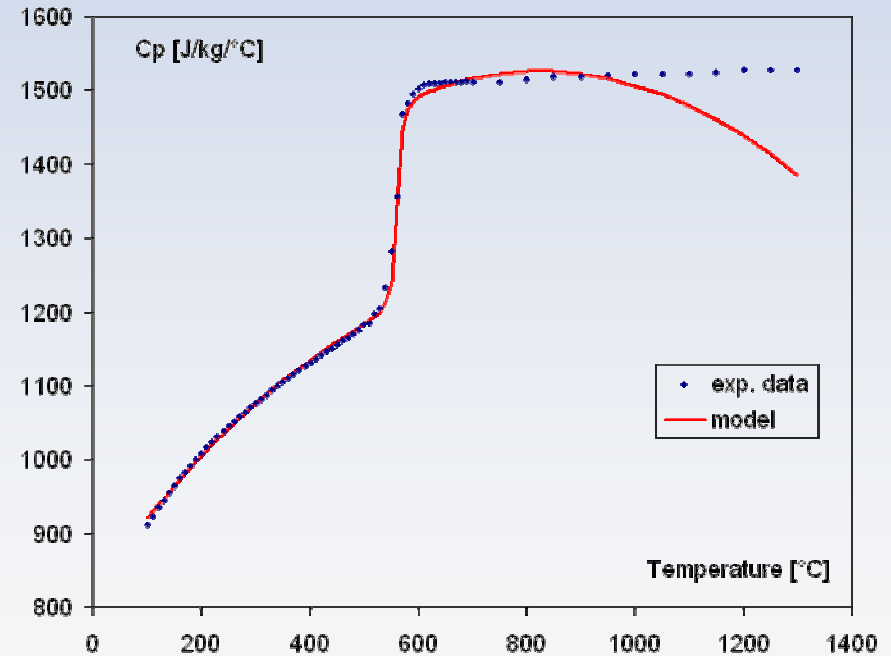
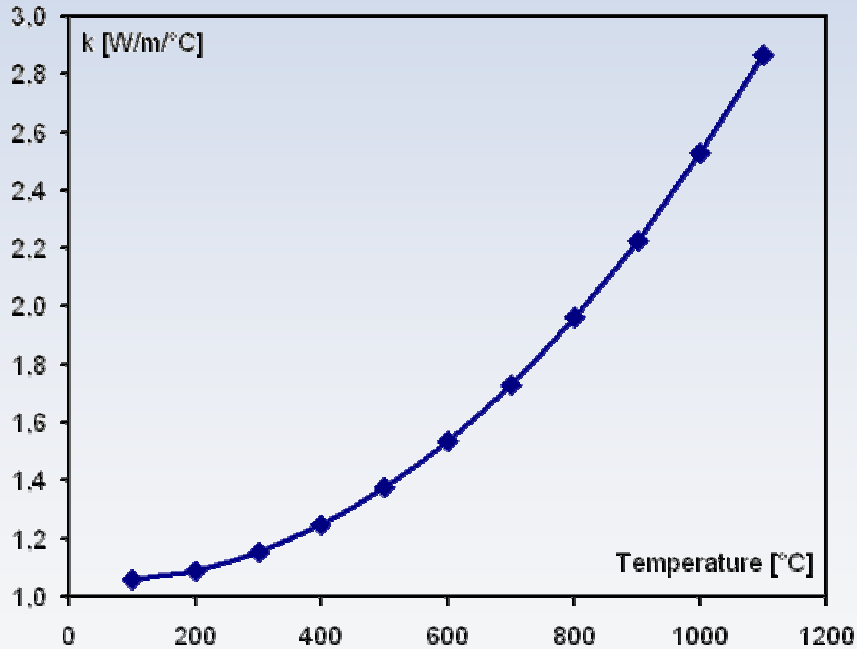
- **Numerical values of the material parameters**
 - Obtaining spectra for glass is expensive, and requires specific equipment;
 - Spectra are measured by Laboratory of Glass Properties LLC, www.glass-labs.com Sankt Peterburg, Russia [formerly Thermex Corp.];
 - Time constants typically range from 1 to 10^3 s at high temperature

- **Numerical values of the material parameters**

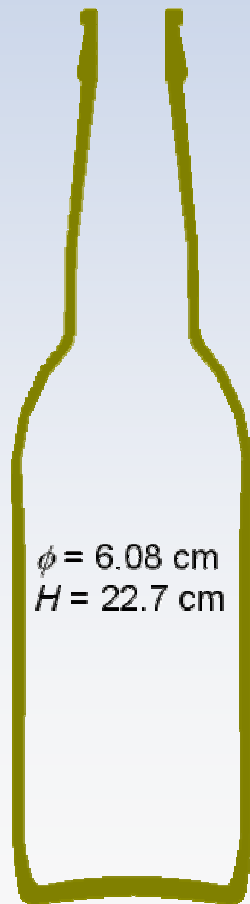
- Other data

- $\rho = 2,445 \text{ kg/m}^3$
- $k = 1.0586 - 0.139 \cdot 10^{-3} T - 1.493 \cdot 10^{-6} T^2 - 0.1146 \cdot 10^{-9} T^3 \text{ W/m/K}$
- $C_p = 824 + 1.0235 T - 0.621 \cdot 10^{-3} T^2 \text{ J/kg/K}$
+ Heaviside function...
- $a_g = 9.5 \cdot 10^{-6}$, $a_l = 30 \cdot 10^{-6}$
- $x = 0.58$, $A = 74,143 \text{ K}$, $T_{\text{ref}} = 553.1 \text{ K}$

- Thermal conductivity and heat capacity



- Geometry, cooling conditions, with blown air

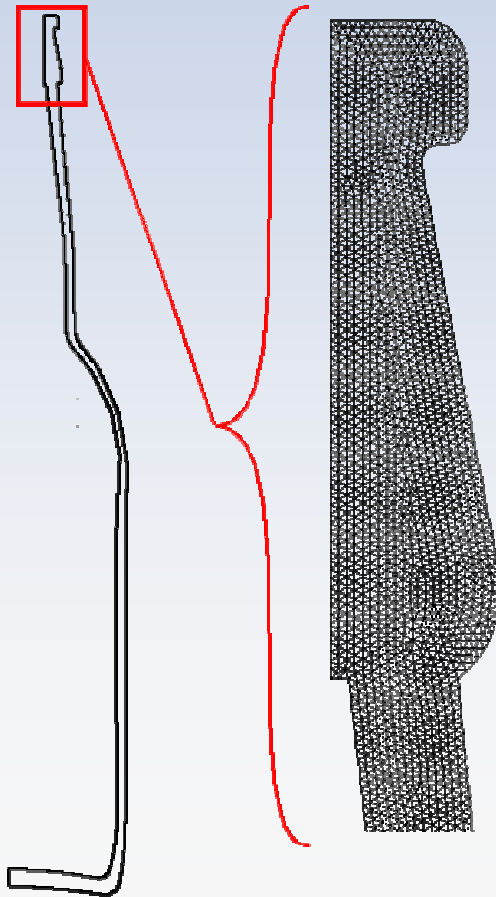


Air temperature: $\sim 27^\circ\text{C}$

Heat transfer coefficient:

$$\alpha_{\text{in}} / \alpha_{\text{out}} = 4/5$$

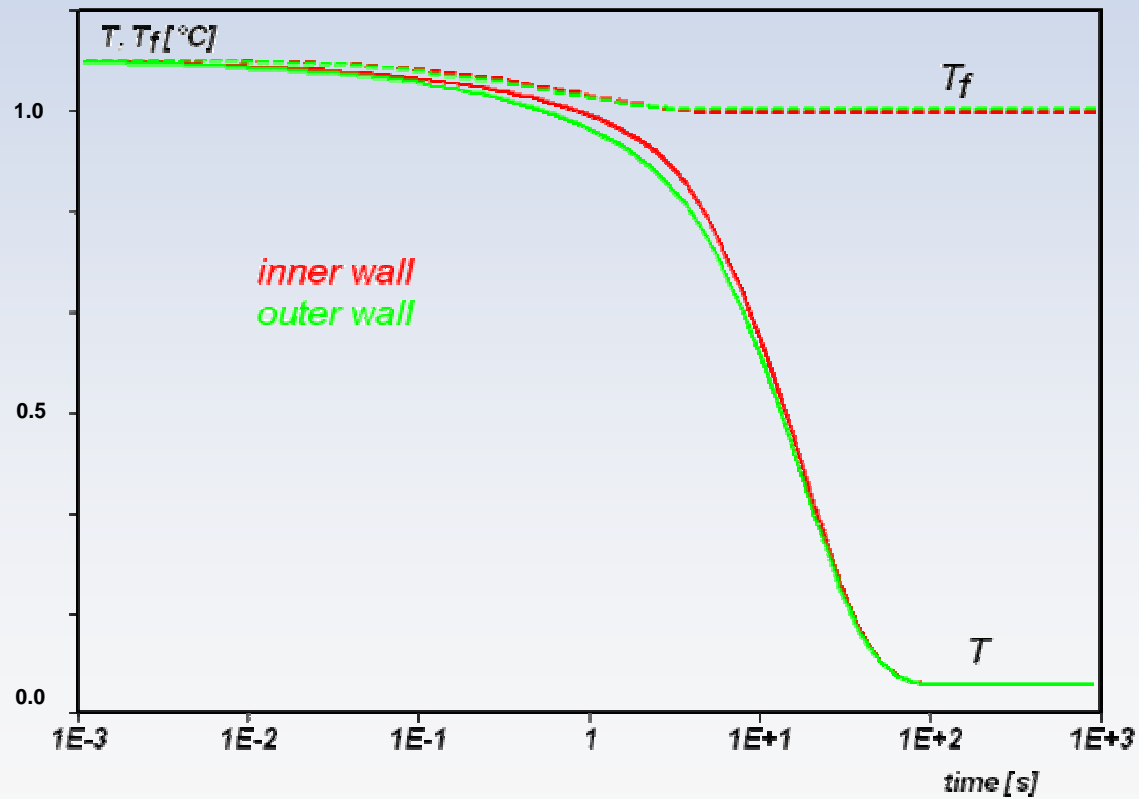
- **Finite element discretisation**



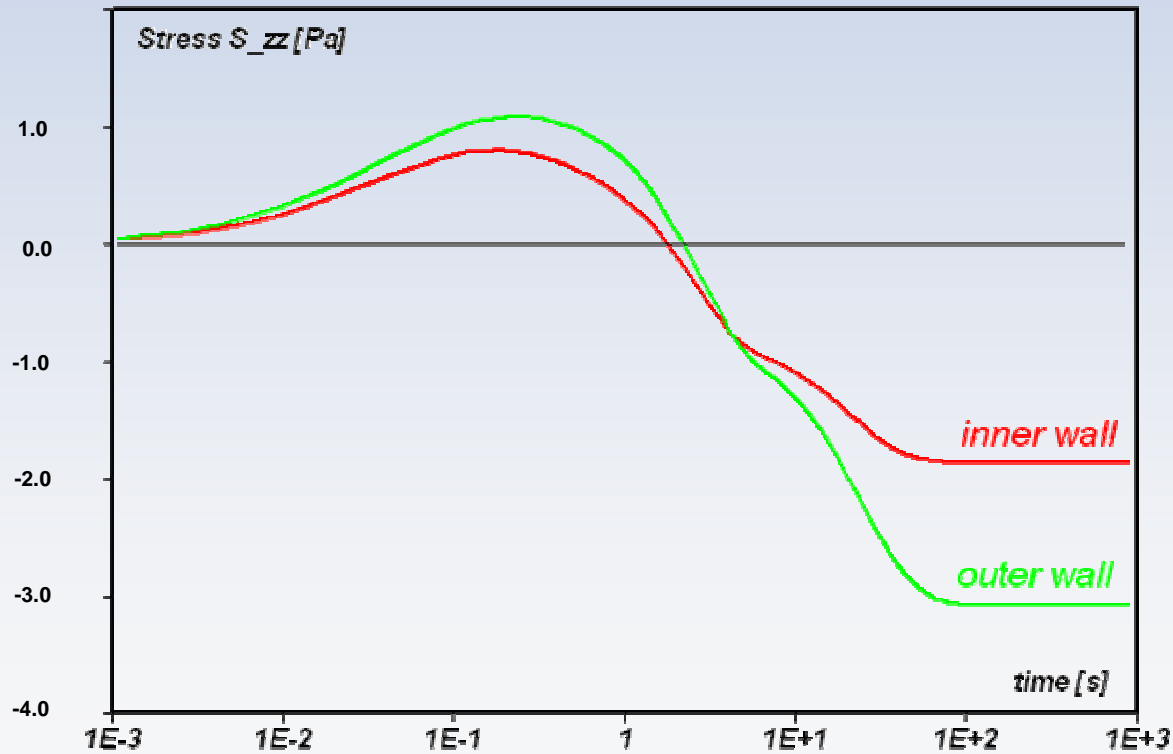
(sample, at the neck)

Total:
24,396 elements
50,865 nodes for T

- **Simulation 1: blown air cooling**



- **Simulation 1: blown air cooling**



- **Measurements of surface stress**

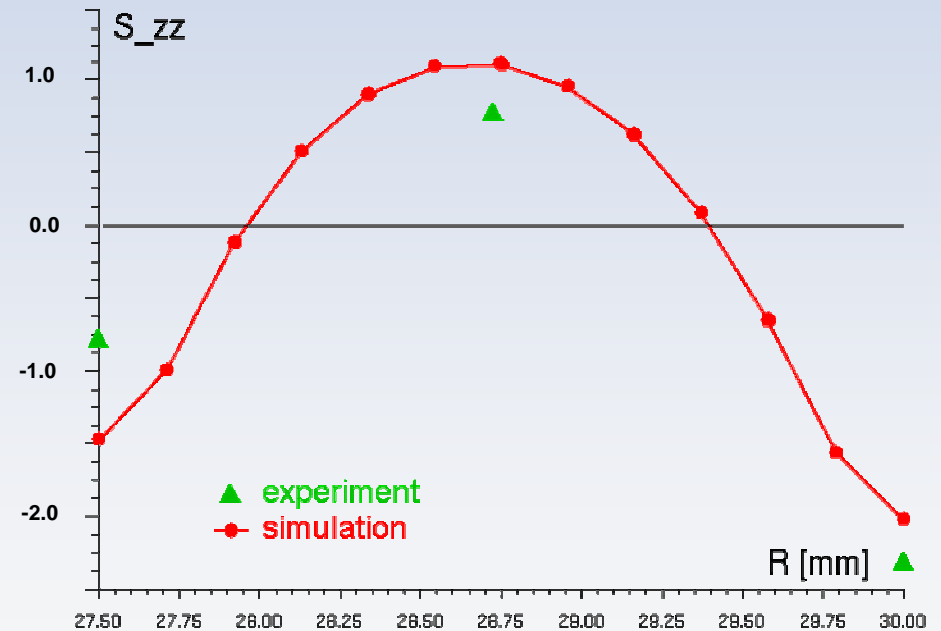
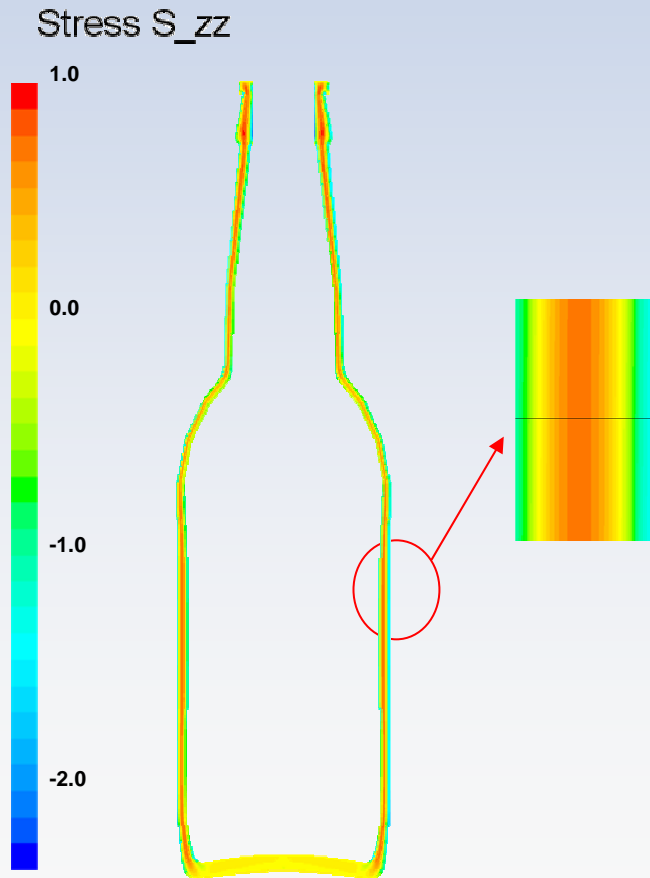
- Equipment used

- Automatic polariscope AP-02 SM
[Laboratory of Photoelasticity
Institute of Cybernetics at TTU, Tallinn, Estonia]
- based on birefringence

- Measurements indicate about:

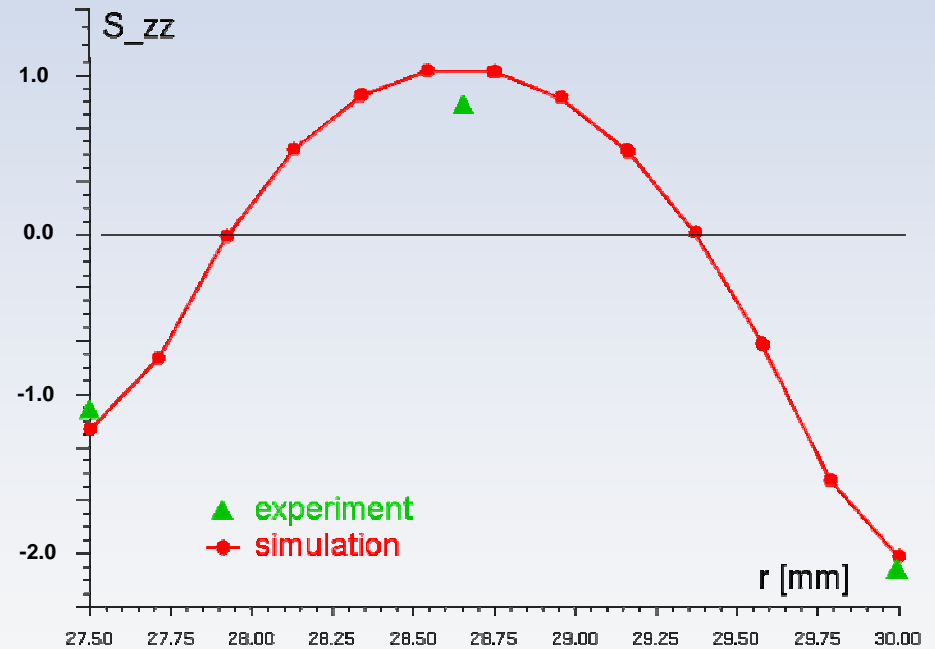
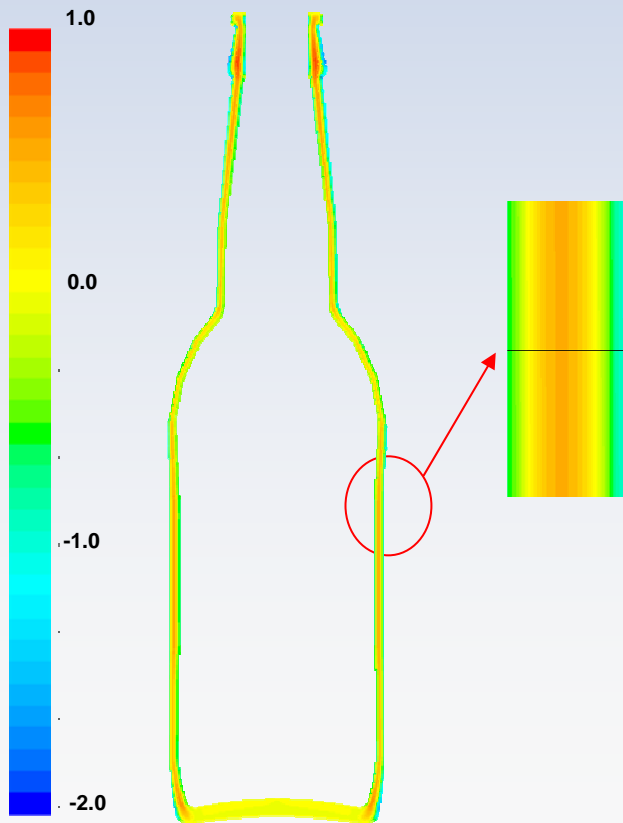
Time [s]	Non-dimensionalized S_{zz} component [Mpa]		
	Inner surface	Centre	Outer surface
20	-0.71	0.62	-2.14
10^4	-1.52	1.00	-2.95

- **Simulation 1: blown air cooling, at t = 20 s:**



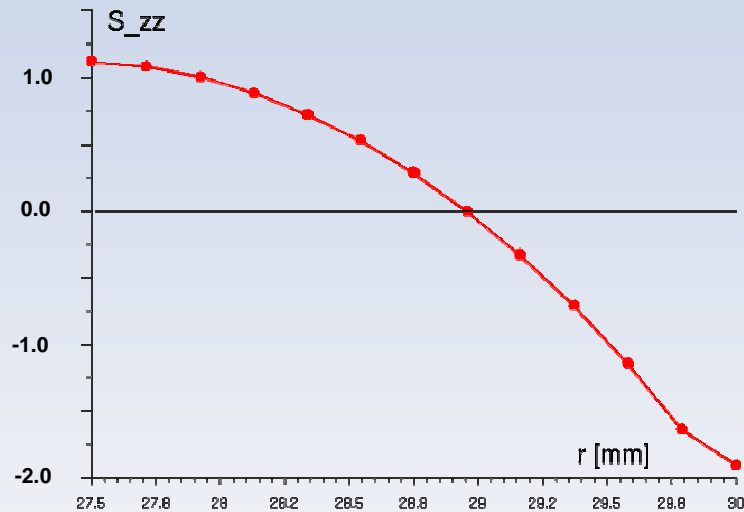
- **Simulation 1: blown air cooling, at t = 10⁴ s:**

Stress S_{zz}



- **departure w/r to experiments at 20s:**
 - uncertainty on the cooling parameters
 - uncertainty on the distribution of cooling
 - calibration issues
 - short time interval for measurement

- **Simulation 2: outside radiation only**



- Occurrence of positive stress at the inner surface of the bottle
- Cause of defect (risk of fracture)
- Operating condition to reject

- **Observations from the physics**
 - Cooling needed on both inner and outer surfaces
 - Prediction of compression state after 20 s
 - Results in agreement with experimental data
- **Observation from the computation**
 - Calculation time: 1.5 h on Dell SC1435
Processor: AMD 2220 SE 2.8GHz
 - 61 steps for 20 s of experimental time
 - [193 steps for 10^4 s of experimental time]

- **Implementation of the Narayanaswamy model**
- **Validation vs. existing results**
- **Application to an industrially relevant case**
- **Comparison with experimental data**
- **Challenge of getting material data**

- **Filling the gap between glass forming and structural modeling**
 - Seamless interface between ANSYS POLYFLOW and ANSYS Mechanical

- **The model**

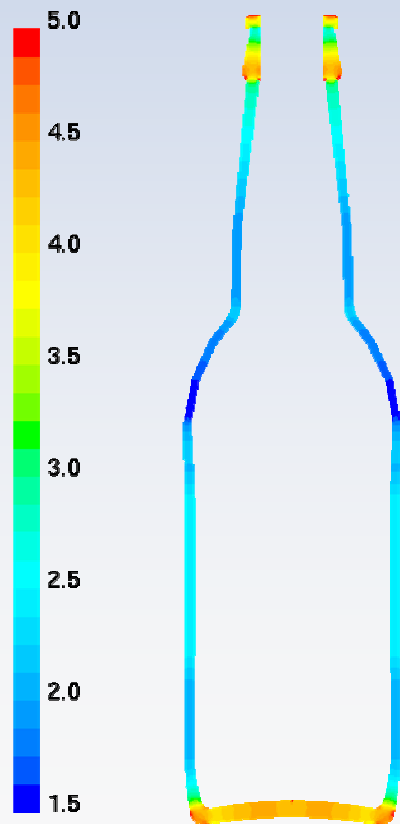
- R.S. Chambers, Numerical integration of the hereditary integrals in a viscoelastic model for glass, J. Am. Ceram. Soc., 75 (1992) 2213-2218 Third level text
- O. S. Narayanaswamy, A Model of Structural Relaxation in Glass, J. Am. Ceram. Soc., 54 (1971) 491-498
- A. Markovsky and T.F. Soules, An efficient and stable algorithm for calculating fictive temperatures, J. Am. Ceram. Soc., 67 (1984) C-66-C-67

- **The software implementation**

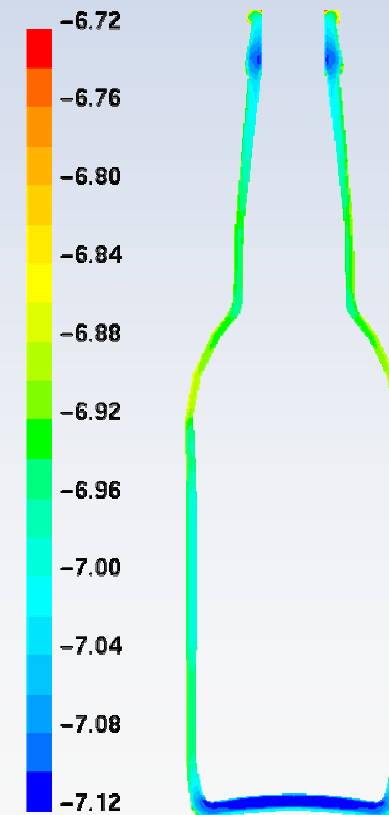
- POLYFLOW version 3.12, user's manual, ANSYS Inc, Canonsburg, PA

- **Simulation 1: blown air cooling**

Thickness [mm]



Thermal strain \ominus * 1000 [-]

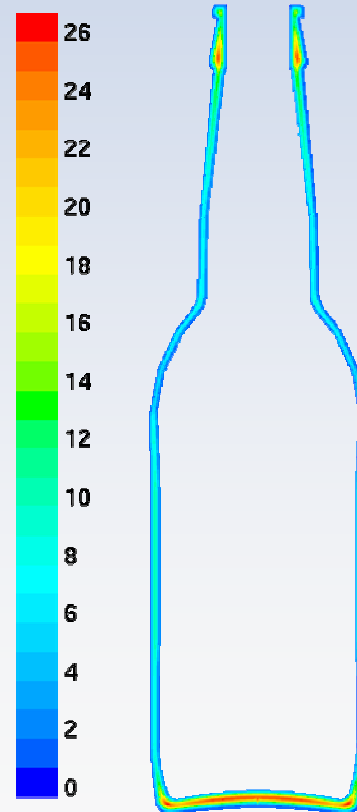


- **Simulation 1: blown air cooling**

Fictive temperature [°C]

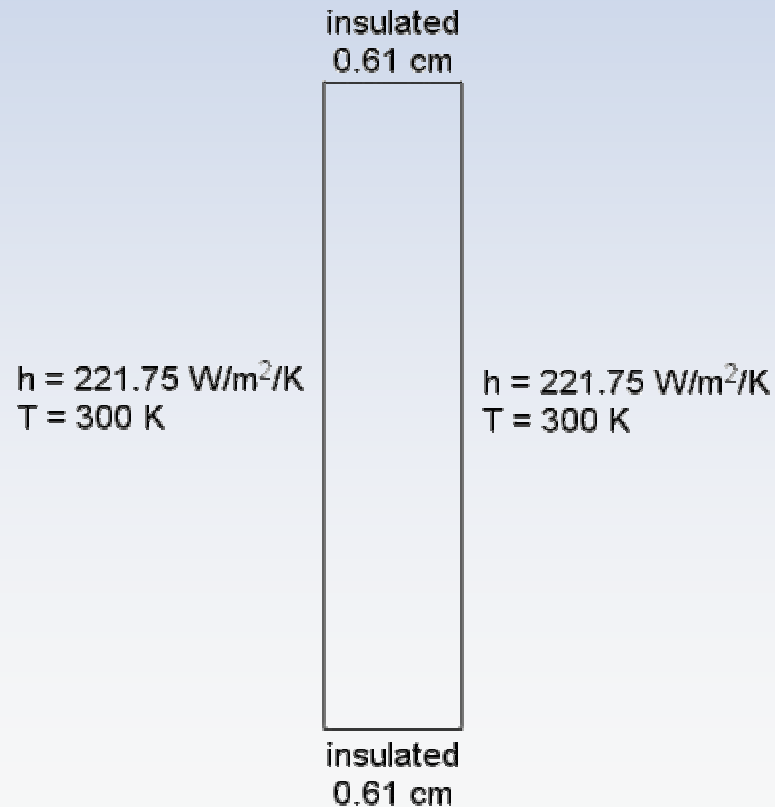


Reduced time ξ [10^3 s]

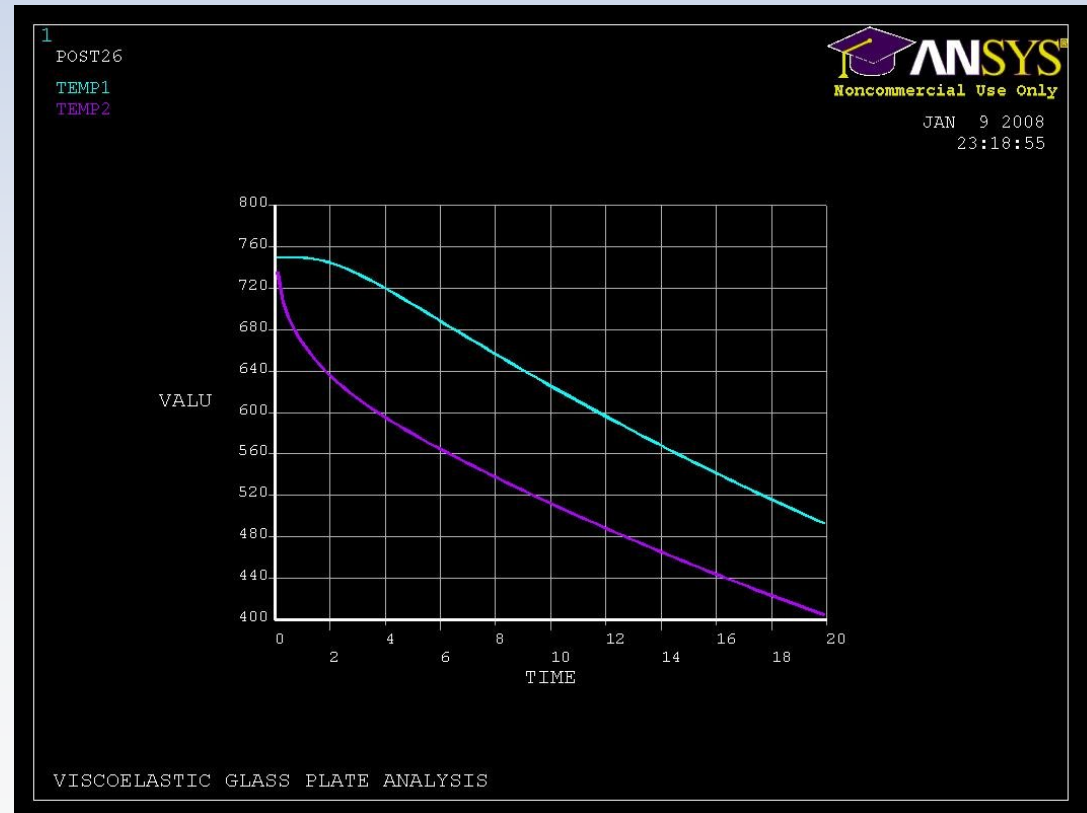
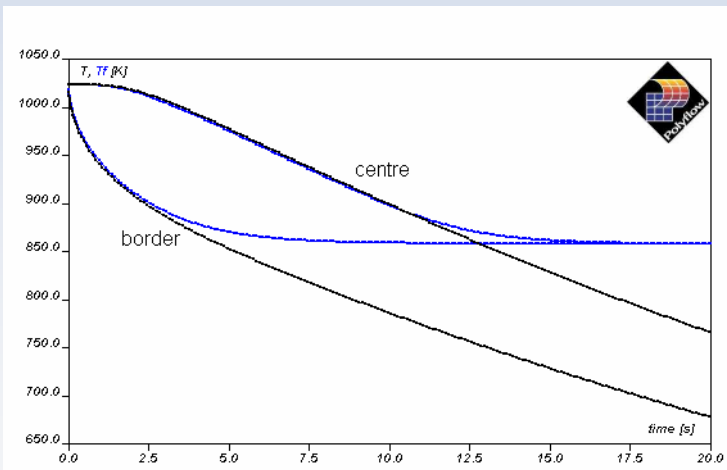


- **Approach suggested by Chambers et al.**
 - Time marching scheme
 - Calculation of temperature field
 - Calculation of fictive temperature and auxiliary fields
 - Calculation of thermal strain
 - Calculation of stresses and deformation

- **Cooling of a plate: definition**



- **Cooling of a plate: temperature calculations**



- **Cooling of a plate: stress calculations**

